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APPLICATION NO.	FILING DATE	FIRST NAMED INVENTOR	ATTORNEY DOCKET NO.	CONFIRMATION NO.
10/771,843	02/04/2004	LeNoir E. Zaiser	2173.2007-001	9748
59242 7590 05/25/2010 R.D. JOHNSON & ASSOCIATES, P.C. 20 PICKERING STREET P.O. BOX 920353 NEEDHAM, MA 02492				
EXAMINER WEINSTEIN, LEONARD J				
ART UNIT 3746		PAPER NUMBER		
NOTIFICATION DATE 05/25/2010		DELIVERY MODE ELECTRONIC		

Please find below and/or attached an Office communication concerning this application or proceeding.

The time period for reply, if any, is set in the attached communication.

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Office Action Summary

Application No.

10/771,843

Applicant(s)

ZAISER ET AL.

Examiner

LEONARD J. WEINSTEIN

Art Unit

3746

Period for Reply -- The MAILING DATE of this communication appears on the cover sheet with the correspondence address --

A SHORTENED STATUTORY PERIOD FOR REPLY IS SET TO EXPIRE 3 MONTH(S) OR THIRTY (30) DAYS, WHICHEVER IS LONGER, FROM THE MAILING DATE OF THIS COMMUNICATION.

- Extensions of time may be available under the provisions of 37 CFR 1.136(a). In no event, however, may a reply be timely filed after SIX (6) MONTHS from the mailing date of this communication.
- If NO period for reply is specified above, the maximum statutory period will apply and will expire SIX (6) MONTHS from the mailing date of this communication.
- Failure to reply within the set or extended period for reply will, by statute, cause the application to become ABANDONED (35 U.S.C. § 133). Any reply received by the Office later than three months after the mailing date of this communication, even if timely filed, may reduce any earned patent term adjustment. See 37 CFR 1.704(b).

Status

- 1) ☒ Responsive to communication(s) filed on 22 March 2010.
- 2a) ☐ This action is **FINAL**. 2b) ☒ This action is non-final.
- 3) ☐ Since this application is in condition for allowance except for formal matters, prosecution as to the merits is closed in accordance with the practice under *Ex parte Quayle*, 1935 C.D. 11, 453 O.G. 213.

Disposition of Claims

- 4) ☒ Claim(s) 3,9-15, 19-24, 27, 33-39, 43-58, 60, 62, 63, 65, 66 and 68 is/are pending in the application.
- 4a) Of the above claim(s) _____ is/are withdrawn from consideration.
- 5) ☐ Claim(s) _____ is/are allowed.
- 6) ☒ Claim(s) 3,9-15, 19-24, 27, 33-39, 43-58, 60, 62, 63, 65, 66 and 68 is/are rejected.
- 7) ☐ Claim(s) _____ is/are objected to.
- 8) ☐ Claim(s) _____ are subject to restriction and/or election requirement.

Application Papers

- 9) ☐ The specification is objected to by the Examiner.
- 10) ☐ The drawing(s) filed on _____ is/are: a) ☐ accepted or b) ☐ objected to by the Examiner.
Applicant may not request that any objection to the drawing(s) be held in abeyance. See 37 CFR 1.85(a).
Replacement drawing sheet(s) including the correction is required if the drawing(s) is objected to. See 37 CFR 1.121(d).
- 11) ☐ The oath or declaration is objected to by the Examiner. Note the attached Office Action or form PTO-152.

Priority under 35 U.S.C. § 119

- 12) ☐ Acknowledgment is made of a claim for foreign priority under 35 U.S.C. § 119(a)-(d) or (f).
- a) ☐ All b) ☐ Some * c) ☐ None of:
1. ☐ Certified copies of the priority documents have been received.
 2. ☐ Certified copies of the priority documents have been received in Application No. _____.
 3. ☐ Copies of the certified copies of the priority documents have been received in this National Stage application from the International Bureau (PCT Rule 17.2(a)).

* See the attached detailed Office action for a list of the certified copies not received.

Attachment(s)

- 1) ☒ Notice of References Cited (PTO-892)
- 2) ☐ Notice of Draftsperson's Patent Drawing Review (PTO-840)
- 3) ☐ Information Disclosure Statement(s) (PTO/SB/08)
Paper No(s)/Mail Date _____
- 4) ☐ Interview Summary (PTO-413)
Paper No(s)/Mail Date _____
- 5) ☐ Notice of Informal Patent Application
- 6) ☐ Other: _____

DETAILED ACTION

1. A request for continued examination under 37 CFR 1.114, including the fee set forth in 37 CFR 1.17(e), was filed in this application after final rejection. Since this application is eligible for continued examination under 37 CFR 1.114, and the fee set forth in 37 CFR 1.17(e) has been timely paid, the finality of the previous Office action has been withdrawn pursuant to 37 CFR 1.114. Applicant's submission filed on March 22, 2010 has been entered.
2. The examiner acknowledges the amendments to claims 3, 11, 15, 21, 27, 35, 39, 45, 49, 54, 60, 65, and 68. The examiner notes that claims 59, 61, 64, and 67 have been canceled.

Claim Rejections - 35 USC § 103

3. The following is a quotation of 35 U.S.C. 103(a) which forms the basis for all obviousness rejections set forth in this Office action:

(a) A patent may not be obtained though the invention is not identically disclosed or described as set forth in section 102 of this title, if the differences between the subject matter sought to be patented and the prior art are such that the subject matter as a whole would have been obvious at the time the invention was made to a person having ordinary skill in the art to which said subject matter pertains. Patentability shall not be negated by the manner in which the invention was made.
4. The factual inquiries set forth in *Graham v. John Deere Co.*, 383 U.S. 1, 148 USPQ 459 (1966), that are applied for establishing a background for determining obviousness under 35 U.S.C. 103(a) are summarized as follows:
 1. Determining the scope and contents of the prior art.
 2. Ascertaining the differences between the prior art and the claims at issue.
 3. Resolving the level of ordinary skill in the pertinent art.
 4. Considering objective evidence present in the application indicating obviousness or nonobviousness.

5. Claims 3, 9-11, 15, 19-21, 33-35, 39, 43-45, 49-58, 60, 62, 63, 65, and 66 are rejected under 35 U.S.C. 103(a) as being unpatentable over Richey, II et al. US 5,988,165 ("Richey") in view of Muratsubaki et al. US 6,068,448 ("Muratsubaki '448"), further in view of Muratsubaki et al. EP 1162372 ("Muratsubaki '372").

a. Independent **claims 15, 39, 49, and 54**

i. **Combination of Richey and Muratsubaki '488**

(1) Statutory Classes

(a) Apparatus of **claims 15 and 49**

(i) **Richey** - Richey teaches all the limitations for a multistage pump for pressurizing a volume of fluid including:

[claims 15 and 49]

a controller (circuitry for compressor 100 shown in figures 4 and 5) in communication with a drive system 105 of a pump/compressor 100 that initiates a piston cycle (piston cycle of pistons 131, 132, and 133) by initiating a compression stroke in a first piston 131 in response to the detection of predetermined pressure within a chamber (buffer tank 200) which corresponds to the pressure in a first piston chamber (chamber defined by cylinder housing element 131 and piston face of piston 131a - see figure 8; Richey - col. 11 ll. 6-29; col. 11 ll. 67-col. 12 ll. 5);

- (ii) **Muratsubaki '448** – Richey does not teach the limitations that are taught by Muratsubaki '448 for a multistage pump for pressurizing fluid including:

[claim 15]

A housing (10A, 10B, 11) having a first cylindrical chamber 12A and a second cylindrical chamber 12B, the first chamber 12A having a first inlet 32 and a first outlet (34/44), the second chamber 12B having a second inlet (34/46) and a second outlet 36, the second inlet (34/46) of the second chamber 12B being in communication with the first outlet (34/44) of the first chamber 12A, a first piston 13A positioned within the first chamber 12A to define a first piston chamber 12A, a second piston 13B positioned within the second chamber 12B to define a second piston chamber 12B the volume of the first piston chamber 12A being larger than the volume of the second piston chamber 12B (col. 9 ll. 62—col. 10 ll. 12), a connecting member (13, 14) for securing the first and second pistons (13A, 13B) together in a spaced apart manner along a common axis (as shown in figure 1), and extending between the first and second chambers (12A, 12B), the connecting member (13, 14) including a threaded portion 13, a drive system (14, 21, 22) for reciprocating the first and second pistons (13A, 13B) in unison within the first and

second piston chambers (12A, 12B) such that when the first piston 13A is moving in an expansion stroke, fluid is drawn into the first piston chamber 12A through the first inlet 32, and at the same time, the second piston 13B is moving in a compression stroke where fluid is expelled from the second piston chamber 12B through the second outlet 36, and when the first piston 13A is moving in a compression stroke, the second piston 13B is moving in an expansion stroke where fluid is expelled from the first piston chamber 12A through the first outlet (34/44) and into the second piston chamber 12B through the second inlet (34/46) where the fluid is compressed due to the reduced volume of the second piston chamber 12B (col. 9 ll. 62 - col. 10 ll. 12), the drive system (14, 21, 22) including a rotatable ball screw nut 14 engaged with the threaded portion 13 and a reversible motor 21 for alternately rotating the nut 14 in opposite directions to cause reciprocating linear translation of the connecting member (13, 14) and pistons, and a check valve system (42, 44, 46, 48) for maintaining a unidirectional flow of fluid from the first inlet 32 to the second outlet 36; and

[claim 49]

A housing (10A, 10B, 11) having an input line 18 for receiving a fluid and an output line 72 for delivering the fluid (col. 9 ll. 62 –

col. 10 ll. 12), a first piston 13A operable in an expansion stroke and a compression stroke in a first piston chamber 12A in the housing (10A, 10B, 11), the first piston chamber 12A having a first inlet 32 in fluid communication with the input line 18 and a first outlet (34/44), wherein during the expansion stroke fluid flows into the first piston chamber 12A through the first inlet 32 and during the compression stroke the fluid is forced out through the first outlet (34/44), a second piston 13B operable in an expansion stroke and a compression stroke in a second piston chamber 12B in the housing (10A, 10B, 11), the second piston chamber 12B having a second inlet (34/46) in fluid communication with the first outlet (34/44) of the first piston chamber 12A and a second outlet 36 in fluid communication with the output line 72, wherein the second piston chamber 12B has a smaller volume than the first piston chamber 12A (col. 9 ll. 62 – col. 10 ll. 12), wherein during the expansion stroke fluid is drawn into the second piston chamber 12B through the second inlet (34/46) and during the compression stroke the fluid is forced out through the second outlet 36, a connecting member (13, 14) securing the first piston 13A and the second piston 13B together in a spaced apart manner along a common axis, as shown in figure 1, the connecting member (13, 14) having

threads (as defined on element 13) along a portion of its length, a ball screw drive system (14, 21, 22) in communication with the threads (as defined on element 13) on the connecting member (13, 14) for reciprocating the connecting member (13, 14) such that when the first piston 13A is in an expansion stroke, the second piston 13B is in a compression stroke, and when the first piston 13A is in a compression stroke, the second piston 13B is in an expansion stroke;

(b) Method of **claims 39 and 54**

- (i) **Richey** - Richey teaches all the limitations for a method of compressing a volume of fluid including the steps of:

[claims 39 and 54]

Initiating piston cycle (piston cycle of pistons 131, 132, and 133) by initiating a compression stroke in a first piston chamber (chamber defined by cylinder housing element 131 and piston face of piston 131a - see figure 8) from a controller (circuitry for compressor 100 shown in figures 4 and 5) in communication with a drive system 105 in response to the detection of a predetermined pressure within the first piston chamber (chamber defined by cylinder housing element 131 and piston face of piston 131a - see figure 8; Richey - col. 11 ll. 6-29; col. 11 ll. 67-col. 12 ll. 5);

- (ii) **Muratsubaki '448** – Richey does not teach the limitations directed toward specific structural components used to perform a method of pressurizing a volume of fluid that are taught by Muratsubaki '448 including the steps of:

[claim 39]

Operating a first piston 13A within a first cylindrical chamber 11A defining a first piston chamber 12A in a housing (10A, 10B, 11), the first piston chamber 12A having a first inlet 32 and a first outlet (34/44), operating a second piston 13B within a second cylindrical chamber 11B defining a second piston chamber 12B in the housing (10A, 10B, 11), the volume of the first piston chamber 12A being larger than the volume of the second piston chamber 12B (col. 9 ll. 62 – col. 10 ll.12), maintaining the first and second pistons (13A, 13B) secured together in a spaced apart manner along a common axis with a connecting member (13, 14), the connecting member (13, 14) including a threaded portion (as defined by the threaded portion of element 13), reciprocating the first and second pistons (13A, 13B) in unison within the first and second piston chambers (12A, 12B) with a drive system (14, 21, 22) such that when the first piston 13A is moving in an expansion stroke, fluid is drawn into the first piston chamber 12A through the first inlet 32, and at

the same time, the second piston 13B is moving in a compression stroke where fluid is expelled from the second piston chamber 12B through the second outlet 36, and when the first piston 13A is moving in a compression stroke, the second piston 13B is moving in an expansion stroke where fluid is expelled from the first piston chamber 12A through the first outlet (34/44) and into the second piston 13B chamber 12B through the second inlet (34/46) where the fluid is compressed due to the reduced volume of the second piston chamber 12B, the drive system (14, 21, 22) including a rotatable ball screw nut 14 engaged with the threaded portion (thread portion of element 13) and a reversible motor 21 for alternately rotating the nut 14 in opposite directions to cause reciprocating linear translation of the connecting member (13, 14) and pistons (13A, 13B) – (col. 10 ll. 34-50), and maintaining a unidirectional flow of fluid from the first inlet 32 to the second outlet 36 with a check vane system (42, 44, 46, 48); and

[claim 54]

The steps of receiving a fluid into a housing (10A, 10B, 11) through an input line 18 and delivering the fluid through an output line 72 (col. 9 ll. 62 – col. 10 ll. 12), operating a first piston 13A in an expansion stroke and a compression stroke in

a first piston chamber 12A in the housing (10A, 10B, 11), the first piston chamber 12A having a first inlet 32 in fluid communication with the input line 18 and a first outlet (34/44), wherein during the expansion stroke fluid flows into the first piston chamber 12A through the first inlet 32 and during the compression stroke the fluid is forced out through the first outlet (34/44), operating a second piston 13B in an expansion stroke and a compression stroke in a second piston chamber 12B in the housing (10A, 10B, 11), the second piston chamber 12B having a second inlet (34/46) in fluid communication with the first outlet (34/44) of the first piston chamber 12A and a second outlet 36 in fluid communication with the output line 72, wherein the second piston chamber 12B has a smaller volume than the first piston chamber 12A, wherein during the expansion stroke fluid is drawn into the second piston chamber 12B through the second inlet (34/46) and during the compression stroke the fluid is forced out through the second outlet 36 (col. 9 ll. 62 – col. 10 ll. 12), securing the first piston 13A and the second piston 13B together with a connecting member (13, 14) in a spaced apart manner along a common axis, the connecting member (13, 14) having threads (as defined by the threaded portion of element 13) along a portion of its length, operating a ball screw drive

system (14, 21, 22) in communication with the threads (as defined by the threaded portion of element 13) on the connecting member (13, 14) to reciprocate the connecting member (13, 14) such that when the first piston 13A is in an expansion stroke, the second piston 13B is in a compression stroke, and when the first piston 13A is in a compression stroke, the second piston 13B is in an expansion stroke.

- (2) **Motivation for combination:** Muratsubaki '448 teaches a pump that is used to create a high pressure by routing a volume of fluid from one chamber where fluid is pressurized by a first piston traversing through a chamber in a compression stroke, to a subsequent second chamber where a second piston that is linked to the first piston traversing through the second chamber in an expansion stroke. When the compression stroke of the first piston is completed a reversible motor drives the first and second piston assembly in a second direction so that the second piston conducts a compression stroke and fluid within the second chamber is pressurized for a second time to a high pressure. Muratsubaki '448 teaches that this method of fluid pressurization eliminates unneeded fluid circulation that can lower the quality of the fluid being pumped or compressed, and reduce the time it takes for fluid to reach a target pressure (Muratsubaki – col. 2 ll. 43-51).

Richey is directed toward a method and apparatus for forming oxygen enriched gas to be store at a high pressure in a mobile storage unit (Richey – Abstract). An objective of Richey is to be able to store high purity oxygen in a pressure vessel using a compressor with a series of pistons that compress a fluid with a first piston which then travels to a chamber of a subsequent piston to be further compressed. Richey utilizes a labyrinth of fluid passages that are used to transport fluid from one piston chamber to the next.

It is noted that in Richey the expansion stroke of each piston does not correspond directly to the compression stroke of any other piston. Muratsubaki '448 minimizes any lost motion of a first piston by using its expansion stroke to further pressurize a fluid volume through its linkage with the second piston. It would have been obvious to one of ordinary skill in the art at the time the invention was made to provide an apparatus and method of pressurizing a volume of fluid, as taught by Richey, with a fluid pump in which a piston assembly with two piston heads is reciprocated by a reversible motor such that the expansion stroke of one piston corresponds the compression stroke of the second piston, as taught by Muratsubaki '448, in order to reduce the risk of negatively affecting the quality of fluid pumped while being able to pressurize the fluid to a desired pressure quickly and more efficiently

by minimizing the lost motion of a pumping member (Muratsubaki '448 – col. 2 ll. 43-51).

ii. **Combination of Richey, Muratsubaki '448, Muratsubaki '372:**

(1) **Muratsubaki '372** - A combination of the references teaches the limitations as discussed but fails to teach the limitations that are taught by Murasukai '372 for:

(a) An apparatus including a pump for pressurizing a volume of fluid including:

[claims 15 and 49]

A drive system (fig. 1) including a reversible motor 7 (analogous to the servo motor 21 of Muratsubaki '448; reversible per Muratsubaki '372 - ¶ [0065]) driving a dual piston pump 1 to reciprocate along a linear path to reciprocate a connecting member (connection between elements 5A and 5B within element 1) and pistons (5A, 5B) wherein the reversible motor 7 has a variable rotational speed (Muratsubaki '372 - ¶ [0067]-[0070]);

(b) A method for operating a pump for pressurizing a volume of fluid including the steps of:

[claims 39 and 54]

Operating a drive system including a reversible motor 7 (analogous to the servo motor 21 of Muratsubaki '448; reversible per Muratsubaki '372 - ¶ [0065]) to drive a dual piston pump 1 to

reciprocate along a linear path thereby reciprocating a connecting member (connection between elements 5A and 5B within element 1) and pistons (5A, 5B) wherein the reversible motor 7 has a variable rotational speed (Muratsubaki '372 - ¶ [0067]-[0070])

- (2) **Motivation for combination:** The plunger pump 10A of Muratsubaki '448 is equivalent to the plunger pump 1 of Muratsubaki '372. The pumps operate in a similar manner with the exception that the pump of Muratsubaki '372 does not teach a valve arrangement that constructively provides an intensifier as taught by Muratsubaki '448. The modification proposed would leave the pump and valve arrangement of Muratsubaki '448 as applied to Richey unchanged but the motor and control of Muratsubaki '448 would be substituted and expanded by the variable speed motor 7 and proportional control module 25 of Muratsubaki '372.

Muratsubaki '372 teaches detecting a pressure value in a nozzle in a suspension state and operating a plunger pump at a maximum rate to quickly raise the actual pressure in the nozzle to a preset value and then continuing to operate the plunger pump at an optimum rate to maintain the pressure value at or very close to the preset value (Muratsubaki '372 - ¶ [0027]). This provides a similar advantage as Muratsubaki '448, however in that pump a threshold pressure can be reached quickly but the speed of the motor driving the pump cannot be

varied continuously to maintain the pressure level dynamically as taught by Muratsubaki '372 (Muratsubaki '372 - ¶ [0068]).

In Richey, while a pressure in the buffer tank is not below a threshold level, the pump is operated until the pressure in a cylinder reaches a threshold high pressure level and then the pump is turned off (Richey - col. 11 ll. 38-45). A modification in which the low pressure cut-off switch of Richey was maintained but the variable speed motor of Muratsubaki '372 was substituted in, and the proportional control module for maintaining a high pressure by varying the speed of a variable speed motor was added to the control of Muratsubaki '448 that senses a piston's position, would teach the limitations as claimed. With this modification once the high pressure level in the cylinder was reached in Richey, just as when the desired pressure is reached in Muratsubaki '372, the control will operate the motor at an optimum rate to maintain that pressure. One of ordinary skill in the art would have found it obvious at the time the invention was made to modify an oxygen concentrator with a mobile oxygen storage unit, as taught by Richey, by substituting a compressor with an intensifier, as taught by Muratsubaki '488 and discussed above, further modified by expanding a control and substituting a motor with a control proportional control module and variable speed motor respectively, as taught by Muratsubaki '372, in order to provide a system that could quickly

provide fluid at a desired pressure in a storage cylinder and dynamically maintain the pressure in the cylinder by modulating the speed at which the motor drove the intensifier (Muratsubaki '372 - ¶ [0027], [0068]).

- b. Dependent **claims 3, 9-11, 19-21, 27, 33-35, 43-45, and 50-58, 60, 62, 63, 65, and 68**

- i. **Richey – claims 10, 20, 34, 44, 50-53, 55-58**

- (1) Apparatus – Richey teaches all the limitations as claimed for pump including:

[claims 10 and 20]

A first pressure sensor (Richey - col. 11 16-19) for sensing fluid pressure in the first piston chamber (chamber defined by cylinder housing element 131 and piston face of piston 131a - see figure 8; Richey - col. 11 ll. 6-29; col. 11 ll. 67-col. 12 ll. 5);

[claims 50 and 52]

Wherein the fluid is a gas (oxygen – Richey – col. 11 ll. 6-8); and

[claims 51 and 53]

Wherein the gas includes concentrated oxygen; (Richey – col. 11 ll. 6-8);

- (2) Method – Richey teaches all the limitations as claimed for method including:

[claims 34 and 44]

The step of sensing the fluid pressure in the first piston chamber (chamber defined by cylinder housing element 131 and piston face of piston 131a - see figure 8; Richey - col. 11 ll. 6-29; col. 11 ll. 67-col. 12 ll. 5) with a first pressure sensor (Richey - col. 11 16-19);

[claims 55 and 57]

The step of the method wherein the fluid is a gas (oxygen – Richey – col. 11 ll. 6-8);

[claims 56 and 58]

The step of the method wherein the gas includes concentrated oxygen; (Richey – col. 11 ll. 6-8);

- ii. **Muratsubaki '488** – The combination of Richey and Muratsubaki '448 involves substituting the entire pump of Richey for the entire pump of Muratsubaki '488, therefore limitations of the following claims are taught by the combination because the pump of Muratsubaki '448 teaches the elements or method steps claimed.

- (1) **Claims 3 and 27 - threaded portion on connecting member**, the pump of Muratsubaki '448 (as applied to the oxygen storage unit of Richey) teaches the limitations including:

- (a) Apparatus -

[claim 3]

A connecting member (13, 14) includes a threaded portion 13, the reversible motor 21 engaging the threaded portion 13, via

elements 14 and 22, for alternately moving the connecting member (13, 14) in opposite directions; and

(b) Method –

[claim 27]

The step wherein the connecting member (13, 14) includes a threaded portion (as defined by the threaded portion of element 13), the reversible motor 21 engaging the threaded portion (threaded portion of element 13), the method further comprising alternately rotating the connecting member (13, 14) in opposite directions with the reversible motor 21 (col. 10 ll. 34-50).

(2) **Claims 9, 19, 33 and 43 – Sensing a piston position**, the pump of Muratsubaki '448 (as applied to the oxygen storage unit of Richey) teaches the limitations including:

(a) Apparatus -

[claims 9 and 19]

A piston position sensing system (col. 14 ll. 4-12) coupled to the drive system (14, 21, 22) to detect when the pistons (13A, 13B) have reached a predetermined stroke and to reverse the drive system (14, 21, 22);

(b) Method -

[claims 33 and 43]

The step of sensing piston position with a piston position sensing system (col. 14 ll. 4-12);

- (3) **Claims 11, 21, 35, and 45 – Second pressure sensor**, the pump of Muratsubaki '448 (as applied to the oxygen storage unit of Richey) teaches the limitations including:

- (a) Apparatus -

[claims 11 and 21]

A pressure sensor 64 for sensing the pressure of fluid expelled from the second piston chamber 12B;

- (b) Method -

[claim 35 and 45]

The step of sensing pressure of fluid expelled from the second piston chamber 12B with a pressure sensor 64;

- (4) **Claims 63 and 66 – Plurality of check valves**, the pump of Muratsubaki '448 (as applied to the oxygen storage unit of Richey) teaches the limitations including:

- (a) Apparatus -

[claim 63]

A plurality of check valves (42, 46, 48) for maintaining a unidirectional flow of fluid from the first inlet 32 to the second outlet 36. A pressure sensor 64 for sensing the pressure of fluid expelled from the second piston chamber 12B;

(b) Method -

[claim 66]

The step of maintaining unidirectional flow of fluid from the first inlet 32 to the second outlet 36 using a plurality of check valves (42, 46, 48).

- iii. **Muratsubaki '372** – The combination of Richey, Muratsubaki '448, and Muratsubaki '372 involves substituting the motor of Muratsubaki '488 with variable motor of Muratsubaki '372 and expanding the control of Muratsubaki '488 with proportional control module of Muratsubaki '372 for operating a variable speed motor at an optimum rate that will vary once a pressure level is reached as a result of system conditions.

(1) **Claims 60, 62, 65, and 68 - Varying speed during piston cycle -**

The proportional control module of Muratsubaki '372 enables dynamic control of the speed of the motor dependent upon system conditions while the motor is drives a plunger pump (Muratsubaki '372 - ¶ [0068]). The dynamic control Muratsubaki will result the speed of the motor changing while a piston is in an intermediate portion of its stroke. Therefore the combination, with respect to the contributions of the proportional control module of Muratsubaki '372, teaches the limitations for a pump including:

(a) Apparatus -

[claims 60 and 65]

Wherein the rotation speed of a motor 7 varies during a piston cycle (Muratsubaki '372- ¶0068);

(b) Method –

[claims 62 and 68]

The step wherein the rotation speed of a motor 7 is varies during a piston cycle (Muratsubaki '372- ¶0068).

6. Claims 12-13, 22-23, 36-37, and 46-47 are rejected under 35 U.S.C. 103(a) as being unpatentable over Richey view of Muratsubaki '448 and Muratsubaki '372, as applied to claims 15, 39, 49, and 54 above in section 5 of this office action. A combination of the references teaches the general conditions of the claimed invention except for the express disclosure of an apparatus for accomplishing a method of compressing fluid including: **[claims 12, 22, 36, and 46]** a ratio of the volume of a first piston chamber to the volume of a second piston chamber is about 12.5 to 1.0; **[claims 13, 23, 37, and 47]** and first and second pistons have a stroke of about 6 inches. It would have been obvious to one having ordinary skill in the art at the time the invention was made to alter the ratio between first and second pumping chambers to be in the range of 12.5 to 1 and a piston stroke being about six inches, since the claimed values are merely an optimum or workable range. It has been held that where the general conditions of a claim are disclosed in the prior art, discovering the optimum or workable ranges involves only routine skill in the art. In re Aller, 105 USPQ 233.
7. Claims 14, 24, 38, and 48 are rejected under 35 U.S.C. 103(a) as being unpatentable over Richey view of Muratsubaki '448 and Muratsubaki '372, as applied

above to claims 13, 24, 37, and 47 in section 6 of this action. A combination of the references teaches the claimed invention except for the limitation of **[claims 14, 24, 38, and 48]** a pump being capable of pumping about 0.5 in.3 of gas at about 2200 psi per piston cycle. The volume of a discharged portion of fluid from a pump and the pressure at which it is discharged is a results effective variable with the results being 0.5 in.3 of gas at about 2200 psi per piston cycle. It would have been obvious to one having ordinary skill in the art at the time the invention was made to provide a pump that was capable of pumping 0.5 in.3 of gas at about 2200 psi per piston cycle, since it has been held that discovering an optimum value of a result effective variable involves only routine skill in the art. *In re Boesch*, 617 F.2d 272, 205 USPQ 215 (CCPA 1980).

Response to Arguments

8. Applicant's arguments with respect to claims 3, 9-15, 19-24, 27, 33-39, and 43-58, 60, 62, 63, 65, 66, and 68 have been considered but are moot in view of the new ground(s) of rejection.

Conclusion

Any inquiry concerning this communication or earlier communications from the examiner should be directed to LEONARD J. WEINSTEIN whose telephone number is (571)272-9961. The examiner can normally be reached on Monday - Thursday 7:00 - 5:30.

If attempts to reach the examiner by telephone are unsuccessful, the examiner's supervisor, Devon Kramer can be reached on (571) 272-7118. The fax phone number for the organization where this application or proceeding is assigned is 571-273-8300.

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